## Exercises for Section 5.2, Exercise 2 (p. 181)

For this situation, the stress-strain pairs are:

$$(\sigma_x = 0, \epsilon_x = ?); (\sigma_y = -25\text{MPa}, \epsilon_y = ?); (\tau_{xy} = ?, \gamma_{xy} = 0)$$

The stress-strain relations in terms of the transformed, or off-axis, reduced compliances, eq. 5.25, become,

$$\begin{bmatrix} \varepsilon_{x} \\ \varepsilon_{y} \\ 0 \end{bmatrix} = \begin{bmatrix} \overline{S}_{11} & \overline{S}_{12} & \overline{S}_{16} \\ \overline{S}_{12} & \overline{S}_{22} & \overline{S}_{26} \\ \overline{S}_{16} & \overline{S}_{26} & \overline{S}_{66} \end{bmatrix} \begin{bmatrix} 0 \\ \sigma_{y} \\ \tau_{xy} \end{bmatrix}$$

The third equation becomes

$$\tau_{xy} = -\left(\frac{\overline{S}_{26}}{\overline{S}_{66}}\right)\sigma_{y}$$

And using this in the first and second equations,

$$\varepsilon_x = \left(\overline{S}_{12} - \frac{S_{16}S_{26}}{\overline{S}_{66}}\right)\sigma_y$$

$$\varepsilon_y = \left(\overline{S}_{22} - \frac{\overline{S}_{26}\overline{S}_{26}}{\overline{S}_{66}}\right)\sigma_y$$

where it is understood that the transformed reduced compliances are evaluated at  $\theta$ =30°. Using numerical values,

$$\bar{S}_{11} = 50.8 (\text{TPa})^{-1}$$
  $\bar{S}_{22} = 88.9$   
 $\bar{S}_{12} = -26.9$   $\bar{S}_{26} = -3.77$   
 $\bar{S}_{16} = -62.2$   $\bar{S}_{66} = 126.0$ 

The answers to parts (a) and (b) are:

$$\varepsilon_x = 719 \mu \varepsilon$$

$$\varepsilon_y = -2220 \mu \varepsilon$$

$$\tau_{xy} = -0.748 \text{MPa}$$

The element expands in the x direction and contracts in the y direction, and a small negative shear stress is required to enforce the shear constraint.

## Exercises for Section 5.3, Exercise 2 (p. 191)

For this situation, the stress-strain pairs are:

$$(\sigma_x = ?, \epsilon_x = -0.050/50.); (\sigma_y = ?, \epsilon_y = -0.050/50.); (\tau_{xy} = ?, \gamma_{xy} = 0)$$

The stress-strain relations in terms of the transformed reduced stiffnesses, eq. 5.83, become,

$$\begin{bmatrix} \sigma_{x} \\ \sigma_{y} \\ \tau_{xy} \end{bmatrix} = \begin{bmatrix} \overline{Q}_{11} \ \overline{Q}_{12} \ \overline{Q}_{16} \\ \overline{Q}_{12} \ \overline{Q}_{22} \ \overline{Q}_{26} \\ \overline{Q}_{16} \ \overline{Q}_{26} \ \overline{Q}_{66} \end{bmatrix} \begin{bmatrix} \varepsilon_{x} \\ \varepsilon_{y} \\ 0 \end{bmatrix}$$

where

$$\varepsilon_x = \varepsilon_y = \frac{-0.050}{50} = -0.001000$$

Using numerical values, as in eq. 5.97,

$$\overline{Q}_{11} = 92.8 \text{GPa}$$
  $\overline{Q}_{22} = 21.0$   $\overline{Q}_{12} = 30.1$   $\overline{Q}_{26} = 15.47$   $\overline{Q}_{16} = 46.7$   $\overline{Q}_{66} = 31.5$ 

and the required stresses are

$$\sigma_x = -122.9 \text{MPa}$$
  $\sigma_y = -51.1$   $\tau_{xy} = -62.2$ 

## Exercise for Section 5.7, Exercise 2 (p. 211)

For this situation, the stress-strain pairs are:

$$(\sigma_x =?, \epsilon_x =0); (\sigma_y =0, \epsilon_y =?); (\tau_{xy} =0, \gamma_{xy} =?)$$

The stress-strain relations, including free thermal strain effects, in terms of the compliances, 65.160 with  $\Delta M = 0$ , become,

$$\begin{bmatrix} 0 - \alpha_x \Delta T \\ \varepsilon_y - \alpha_y \Delta T \\ \gamma_{xy} - \alpha_{xy} \Delta T \end{bmatrix} = \begin{bmatrix} \overline{S}_{11} \ \overline{S}_{12} \ \overline{S}_{16} \\ \overline{S}_{12} \ \overline{S}_{22} \ \overline{S}_{26} \\ \overline{S}_{16} \ \overline{S}_{26} \ \overline{S}_{66} \end{bmatrix} \begin{bmatrix} \sigma_x \\ 0 \\ 0 \end{bmatrix}$$

**Expanding** 

$$-\alpha_x \Delta T = \overline{S}_{11} \sigma_x$$

$$\varepsilon_y - \alpha_y \Delta T = \overline{S}_{12} \sigma_x$$

$$\gamma_{xy} - \alpha_{xy} \Delta T = \overline{S}_{16} \sigma_x$$

Solving the first stress-strain relation for  $\sigma_x$  , and then the second and third relations for  $\epsilon_x$  and leads to

$$\sigma_{x} = \frac{-\alpha_{x}\Delta T}{\overline{S}_{11}} \qquad \varepsilon_{y} = \left(\alpha_{y} - \frac{\overline{S}_{12}}{\overline{S}_{11}}\alpha_{x}\right)\Delta T \qquad \gamma_{xy} = \left(\alpha_{xy} - \frac{\overline{S}_{16}}{\overline{S}_{11}}\alpha_{x}\right)\Delta T$$

Also for this case, from eq. 5.165,

$$\varepsilon_z = \alpha_z \Delta T + (S_{13} \cos^2 \theta + S_{23} \sin^2 \theta) \sigma_z$$

The material properties for  $\theta = 45^{\circ}$  are:

$$\bar{S}_{11} = 78.3 (\text{TPa})^{-1}$$
  $\bar{S}_{12} = -35.3$   $\bar{S}_{16} = -38.1$   $\alpha_x = 12.14x10^{-6}/^{\circ}C$   $\alpha_y = 12.14x10^{-6}$   $\alpha_{xy} = -24.3x10^{-6}$ 

As a result

$$\sigma_x = -7.75 \text{MPa}$$
  $\varepsilon_y = 881 \times 10^{-6}$   $\gamma_{xy} = -0.000921 (-0.0527^{\circ})$   $\varepsilon_z = 1368 \times 10^{-6}$ 

Since  $\theta = 45^{\circ}$ , this problem and the problem of fig. 5.15 are practically identical, namely, a constraint in the direction perpendicular to the applied compressive stress, and the fibers making a 45° angle with that stress.